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# Buckling of Stepped Thickness Plates in the Theory of Plasticity

The problem of buckling has been treated for a long time as a single rectangular field with different boundary conditions loaded with inplate loads and usualy applying Levy method, when two opposite edges were always simply supported. This paper is concerned with the plastic buckling of rectangular plates in the region of plasticity. In this paper, it has been treated the elasto-plastic deformations for two coupled plates with different thicknesses, loaded with inplane constant forces  $N_x = \text{const.}$ 

Keywords: buckling, plates, plasticity, deformations.

## 1. INTRODUCTION

The problem of buckling has been treated for many years as one rectangular field with different boundary conditions loaded with in-plate loads  $N_x$ , and  $N_y$  and usually applying Levy method, when two opposite edges were always simply supported.

In plastic region A. A. Ilyushin [1] and E. Z. Stowell [2], using their differential equations solved many buckling problems in elasto-plastic region, but always with plates of constant thickness. Now, in this paper, the author has solved the problem of buckling when two rectangular plates form the unity, but of different thicknesses. All sides are simply supported and pressed along two parallel opposite sides with  $N_x = \text{const.}$ 

It is known, Fig. 1, that in the plastic region the angle  $\alpha_0$  corresponds to modulus of elasticity (*E*), elastic region, and direction ON<sub>0</sub>, determined by angle  $\alpha_0$  gives "cutting modulus" of elasticity ( $E_c^0$ ), where the point N<sub>0</sub> gives the initial appearance of plasticity [3]. The value of angle  $\alpha_k$  gives "tangential modulus of plasticity".

$$\mathbf{E}_k^0 = \frac{\mathrm{d}\sigma}{\mathrm{d}\varepsilon}.$$
 (1)



Figure 1. Relation between ( $\alpha$ ,  $\alpha_c$ ,  $\alpha_k$ ) and (E,  $E_c$ ,  $E_k$ )

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The direction ON determines the angle  $\alpha_c$  which determines primary modulus of elasticity

$$E_c^0 = \frac{\sigma}{\varepsilon} \,. \tag{2}$$

According to the hypothesis that the radius of the Mises circle may be written, as it is known, as generalized stress, effective stress or equivalent stress.

In the case of complete stresses the intensity of stress  $\sigma_i$  and intensity of deformation  $\varepsilon_i$  are introduced in the forms

$$\sigma_{i} = \frac{1}{\sqrt{2}} \cdot \sqrt{\left(\sigma_{x} - \sigma_{y}\right)^{2} + \left(\sigma_{y} - \sigma_{z}\right)^{2} + \left(\sigma_{z} - \sigma_{x}\right)^{2} + 6\left(\tau_{xy}^{2} + \tau_{yz}^{2} + \tau_{zx}^{2}\right)} (3)$$

and

$$\varepsilon_{i} = \frac{\sqrt{2}}{3} \cdot \sqrt{\left(\varepsilon_{x} - \varepsilon_{y}\right)^{2} + \left(\varepsilon_{y} - \varepsilon_{z}\right)^{2} + \left(\varepsilon_{z} - \varepsilon_{x}\right)^{2} + \frac{3}{2}\left(\gamma_{xy}^{2} + \gamma_{yz}^{2} + \gamma_{zx}^{2}\right)} .$$
(4)

## 2. FORMULATION OF THE PROBLEM

In our case we are solving the problem of stability of plates, when we have small elasto-plastic deformations, using partial differential equations of A. A. Ilyushin [1,2]

$$\left(1 - \frac{3}{4} \frac{1 - s}{1 - r} \frac{\sigma_x^2}{\sigma_i^2}\right) \frac{\partial^4 w}{\partial x^4} + 2 \left(1 - \frac{3}{4} \frac{1 - s}{1 - r} \frac{\sigma_x \sigma_y + 2\tau^2}{\sigma_i^2}\right) \cdot \frac{\partial^4 w}{\partial x^2 \partial y^2} + \left(1 - \frac{3}{4} \frac{1 - s}{1 - r} \frac{\sigma_y^2}{\sigma_i^2}\right) \frac{\partial^4 w}{\partial y^4} - 3 \frac{1 - s}{1 - r} \frac{\tau}{\sigma_i^2} \cdot \left(\sigma_x \frac{\partial^4 w}{\partial x^3 \partial y} + \sigma_y \frac{\partial^4 w}{\partial x \partial y^3}\right) + \frac{1}{(1 - r)D'} \Pi(\sigma, w) = 0.$$
(5)

In (5) is introduced the sign

$$\Pi(\sigma, w) = \sigma_x \frac{\partial^2 w}{\partial x^2} + 2\tau \frac{\partial^2 w}{\partial x \partial y} + \sigma_y \frac{\partial^2 w}{\partial y^2}.$$
 (6)

In our case we consider two plates (1) and (2) (Fig. 2), it is plate of dimensions  $a \times b$  which has contact along the edge  $y = \eta$ ,  $(0 < \eta < b)$ .



Figure 2. Two coupled plates with load  $N_x$  = const.

Now, we have

- plate (1):  $0 \le x \le a$ ;  $0 \le y \le \eta$ •
- plate (2):  $0 \le x \le a$ ;  $\eta \le y \le b$ .
- In (5) are given:  $r = 1 \varphi_c$ ,  $s = 1 (\varphi_c \varphi_k)$ .

Along the edges y = 0 and y = b we could have any kind of boundary conditions. Along the plates (1) and (2), for x = 0 and x = a are pressed with forces  $N_x =$ const. and they have thicknesses  $h_1$  and  $h_2$ . The other forces are  $N_v = 0$  and  $N_{xv} = 0$ .

In actual case is used modified differential equation given by E. Z. Stowell

$$\begin{bmatrix} 1 - \frac{3}{4} \left(1 - \frac{\varphi_k}{\varphi_c}\right) \frac{\sigma_x^2}{\sigma_i^2} \end{bmatrix} \frac{\partial^4 w}{\partial x^4} + 2 \begin{bmatrix} 1 - \frac{3}{4} \left(1 - \frac{\varphi_k}{\varphi_c}\right) \frac{\sigma_x \sigma_y + 2\tau^2}{\sigma_i^2} \end{bmatrix}.$$
$$\cdot \frac{\partial^4 w}{\partial x^2 \partial y^2} + \begin{bmatrix} 1 - \frac{3}{4} \left(1 - \frac{\varphi_k}{\varphi_c}\right) \frac{\sigma_y^2}{\sigma_i^2} \end{bmatrix} \frac{\partial^4 w}{\partial y^4} - 3 \left(1 - \frac{\varphi_k}{\varphi_c}\right).$$
$$\cdot \left(\sigma_x \frac{\partial^4 w}{\partial x^3 \partial y} + \sigma_y \frac{\partial^4 w}{\partial x \partial y^3}\right) + \frac{h}{D'_c} \Pi(\sigma, w) = 0 \quad (7)$$

where values for aluminium and for steel are given in [5].

For our case, given with (Fig. 2), (7) is reduced to:

$$\left(\frac{1}{4} + \frac{3}{4}\frac{\varphi_k}{\varphi_c}\right)\frac{\partial^4 w}{\partial x^4} + 2\frac{\partial^4 w}{\partial x^2 \partial y^2} + \frac{\partial^4 w}{\partial y^4} + \frac{\sigma_x h}{D'_c}\frac{\partial^2 w}{\partial x^2} = 0$$
(8)

 $\varphi_c = \frac{E_c}{E}, \qquad \varphi_k = \frac{E_k}{E}, \qquad D'_c = D'^1_c = D_1$ where or  $D_{0}' = D_{0}'^{2} = D_{2}$ 

d, for the plate (1) and (2) are supposed functions  $w_1$  and  $w_2$ :

$$w_1 = f_1(y) \sin \frac{m\pi x}{a} \tag{9}$$

$$w_2 = f_2(y) \sin \frac{m\pi x}{a}.$$
 (10)

Using (9) and (10), under the load  $N_x = \sigma_x \cdot h$ , the plates are going to deflect in x direction in the form of sin functions, while the functions  $f_1(y)$  and  $f_2(y)$  are giving behaviours of plates (1) and (2) in y direction.

The conditions on edges x = 0 and x = a must be satisfied:

$$w_1 = 0 \Big|_{x=0}$$
 and  $D_1 \left( \frac{\partial^2 w_1}{\partial x^2} + v \frac{\partial^2 w_1}{\partial y^2} \right) = 0 \Big|_{x=a}$  (11)

$$w_2 = 0\Big|_{x=a}$$
 and  $D_2\left(\frac{\partial^2 w_2}{\partial x^2} + v \frac{\partial^2 w_2}{\partial y^2}\right) = 0\Big|_{x=a}$  (12)

which means that plates (1) and (2) are simply supported on x = 0 and x = a, or the deflections and moments are equal to zero.

Supposed solutions must also satisfy the differential equations for plates (1) and (2), respectively:

$$\left(\frac{1}{4} + \frac{3}{4}\frac{\varphi_k}{\varphi_c}\right)\frac{\partial^4 w_1}{\partial x^4} + 2\frac{\partial^4 w_1}{\partial x^2 \partial y^2} + \frac{\partial^4 w_1}{\partial y^4} + \frac{N_x \partial w_1}{D_1 \partial x^2} = 0 \quad (13)$$
$$\left(\frac{1}{4} + \frac{3}{4}\frac{\varphi_k}{\varphi_c}\right)\frac{\partial^4 w_2}{\partial x^4} + 2\frac{\partial^4 w_2}{\partial x^2 \partial y^2} + \frac{\partial^4 w_2}{\partial y^4} + \frac{N_x \partial w_2}{D_2 \partial x^2} = 0 \quad (14)$$

According to obtained solutions for (13) and (14), we have to find the derivations of (9) and (10):

$$w_{1} = f_{1}(y)f_{0}(x) = f_{1}(y)\sin\frac{m\pi x}{a}$$

$$\frac{\partial w_{1}}{\partial x} = f_{1}(y)\frac{m\pi}{a}\cos\frac{m\pi x}{a}; \frac{\partial^{2}w_{1}}{\partial x^{2}}; \frac{\partial^{3}w_{1}}{\partial x^{3}}; \frac{\partial^{4}w_{1}}{\partial x^{4}}$$

$$\frac{\partial^{4}w_{1}}{\partial x^{2}\partial y^{2}} = f_{1}''(y)\left(-\frac{m^{2}\pi^{2}}{a^{2}}\right)\sin\frac{m\pi x}{a}$$

$$\frac{\partial w_{1}}{\partial y} = f_{1}'(y)\sin\frac{m\pi x}{a}; \frac{\partial^{2}w_{1}}{\partial y^{2}}; \frac{\partial^{3}w_{1}}{\partial y^{3}}; \frac{\partial^{4}w_{1}}{\partial y^{4}} \quad (15)$$

$$w_{2} = f_{2}(y)f_{0}(x) = f_{2}(y)\sin\frac{m\pi x}{a}$$

$$\frac{\partial^{4}w_{2}}{\partial x^{2}\partial y^{2}} = f_{2}''(y)\left(-\frac{m^{2}\pi^{2}}{a^{2}}\right)\sin\frac{m\pi x}{a}$$

$$\frac{\partial^{4}w_{2}}{\partial x^{2}\partial y^{2}} = f_{2}''(y)\left(-\frac{m^{2}\pi^{2}}{a^{2}}\right)\sin\frac{m\pi x}{a}$$

$$\frac{\partial w_{2}}{\partial y} = f_{2}'(y)\sin\frac{m\pi x}{a}; \frac{\partial^{2}w_{2}}{\partial y^{2}}; \frac{\partial^{3}w_{2}}{\partial y^{3}}; \frac{\partial^{4}w_{2}}{\partial y^{4}} \quad (16)$$

Using (15) and (16) we obtain:

$$\left(\frac{1}{4} + \frac{3}{4}\frac{\varphi_k}{\varphi_c}\right) f_1(y) \frac{m^4 \pi^4}{a^4} \sin \frac{m\pi x}{a} + 2f_1''(y) \left(-\frac{m^2 \pi^2}{a^2}\right) \sin \frac{m\pi x}{a} + f_1^{IV}(y) \sin \frac{m\pi x}{a} + f_1(y) \left(-\frac{m^2 \pi^2}{a^2}\right) \sin \frac{m\pi x}{a} = 0.$$
(17)

At last from (17) we get:

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$$f_1^{IV}(y) - 2f_1''(y)\frac{m^2\pi^2}{a^2} + f_1(y)\left[\frac{m^4\pi^4}{a^4}\left(\frac{1}{4} + \frac{3}{4}\frac{\varphi_k}{\varphi_c}\right) - \frac{N_x}{D_1}\frac{m^2\pi^2}{a^2}\right] = 0.$$
(18)

Partial differential equation of forth order is reduced to ordinal differential equation with constant coefficients of the same order (18).

The solution for (18) is supposed to be:

$$f_1(y) = e^{\lambda y} . \tag{19}$$

From (18) we obtain characteristic equation

$$\lambda^{4} - 2\frac{m^{2}\pi^{2}}{a^{2}}\lambda^{2} + \left[\frac{m^{4}\pi^{4}}{a^{4}}\kappa - k\frac{\pi^{4}m^{2}}{a^{2}b^{2}}\right]$$
(20)

where  $\kappa_0$  and  $k_1$  are:

$$\kappa_0 = \left(\frac{1}{4} + \frac{3}{4}\frac{\varphi_k}{\varphi_c}\right), \text{ and } k_1 = \frac{N_x b^2}{D_1 \pi^2}.$$
(21)

From (20), we get four solutions for  $\lambda$ :

$$\lambda_{1,2,3,4} = \pm \frac{m\pi}{a} \sqrt{1 \pm \sqrt{\left(1 + k_1 \frac{a^2}{b^2 m^2}\right) - \kappa_0}} \quad (22)$$
  
As it is  $\kappa_0 < 1$ , then  $\left(1 + \frac{k_1 a^2}{b^2 m^2}\right) - \kappa_0 > 0$ .

One possible combination for resolving our problem is:

$$\lambda_{1,2} = \pm \frac{m\pi}{a} \sqrt{1 + \sqrt{\left(1 + k_1 \frac{a^2}{b^2 m^2}\right) - \kappa_0}} = \pm \alpha_1$$
$$\lambda_{3,4} = \pm i \frac{m\pi}{a} \sqrt{-1 + \sqrt{\left(1 + k_1 \frac{a^2}{b^2 m^2}\right) - \kappa_0}} = \beta_1 \cdot i . (23)$$

In this case we obtain our function of deflection:

$$w_1 = (C_1 \operatorname{ch} \alpha_1 y + C_2 \operatorname{sh} \alpha_1 y + C_3 \operatorname{cos} \beta_1 y + C_4 \operatorname{sin} \beta_1 y) \operatorname{sin} \frac{m\pi x}{a}.$$
 (24)

For the plate (2) we have:

$$\lambda^4 - 2\frac{m^2\pi^2}{a^2}\lambda^2 - \left(\frac{N_x}{D_2}\frac{m^2\pi^2}{a^2} - \frac{m^4\pi^4}{a^4}k_1\right) = 0.$$
 (25)

For

$$\frac{N_x}{D_2} \frac{m^2 \pi^2}{a} = \frac{N_x b^2}{D_1 \pi^2} \frac{m^2 \pi^4}{b^2} \frac{D_1}{D_2} = k_1 \frac{D_1}{D_2} \frac{m^2 \pi^4}{b^2} \quad (26)$$

introducing the coefficient  $\psi$ .

$$\psi = \frac{D_1}{D_2}.$$
 (27)

The roots of (25) are

$$\lambda_{1,2,3,4}' = \pm \frac{m\pi}{a} \sqrt{1 \pm \sqrt{\left(1 + k_1 \frac{a^2}{b^2 m^2} \psi\right) - \kappa_0}} \quad (28)$$

or

$$\lambda_{1,2}' = \pm \frac{m\pi}{a} \sqrt{1 \pm \sqrt{\left(1 + k_1 \frac{a^2}{b^2 m^2} \psi\right) - \kappa_0}} = \pm \alpha_2$$
$$\lambda_{3,4}' = \pm i \frac{m\pi}{a} \sqrt{\sqrt{\left(1 + k_1 \frac{a^2}{b^2 m^2} \psi\right) - \kappa_0} - 1} = \pm i\beta_2 . (29)$$

The solution for the plate (2) is now:

$$w_2 = f_2(y) \sin \frac{m\pi x}{a} = \left(C_5 \operatorname{ch} \alpha_2 y + C_6 \operatorname{sh} \alpha_2 y + C_7 \cos \beta_2 y + C_8 \sin \beta_2 y\right) \sin \frac{m\pi x}{a}.$$
 (30)

Now we must use the boundary conditions (I, II, ..., VII, VIII), in order to get unknown coefficients of (24) and (28):

I condition:

$$w_1 = 0 \Big|_{y=0} \tag{31}$$

means that deflection for y = 0 is zero. **II condition:** 

$$\frac{\partial^2 w_1}{\partial y^2} + v \frac{\partial w_1}{\partial x^2} = 0 \bigg|_{y=0}$$
(32)

means that the moment is equal to zero. Using (31) and (32) we get:

I: 
$$(C_1 \operatorname{ch} + C_2 \operatorname{sh} 0 + C_3 \cos 0 + C_4 \sin 0) \sin \frac{m\pi x}{a} = f_1(0) = 0$$
  
 $C_1 + C_3 = 0$   
II:  $w_1 = (C_1 \operatorname{ch} \alpha_1 y + C_2 \operatorname{sh} \alpha_1 y + C_3 \cos \beta_1 y + C_4 \sin \beta_1 y) \sin \frac{m\pi x}{a}$   
 $\frac{\partial w_1}{\partial y} = (\alpha_1 C_1 \operatorname{sh} \alpha_1 y + \alpha_1 C_2 \operatorname{ch} \alpha_1 y - C_3 \beta_1 \sin \beta_1 y + C_4 \beta_1 \cos \beta_1 y) \sin \frac{m\pi x}{a}$   
 $\frac{\partial^2 w_1}{\partial y^2} = (\alpha_1^2 C_1 \operatorname{ch} \alpha_1 y + \alpha_1^2 C_2 \operatorname{sh} \alpha_1 y - C_3 \beta_1^2 \sin \beta_1 y + C_4 \beta_1^2 \cos \beta_1 y) \sin \frac{m\pi x}{a}$ . (33)

Including (33) into (32) we obtain:

$$\left(C_{1}\alpha_{1}^{2} \operatorname{ch0} + C_{2}\alpha_{1}^{2} \operatorname{sh0} - C_{3}\beta_{1}^{2} \cos 0 - C_{4}\beta_{1}^{2} \cdot 0\right) \cdot \\ \cdot \sin \frac{m\pi x}{a} + f_{1}(0) \left(-\frac{m^{2}\pi^{2}}{a^{2}}\nu \sin \frac{m\pi x}{a}\right) = 0. \quad (34)$$

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Taking into account (I),  $f_1(0) = 0$ , condition (II) is reduced to:

$$C_1 \alpha_1^2 - C_3 \beta_1^2 = 0.$$
 (35)

Multiplying by  $\alpha_1^2$ , and subtracting (I) and (II):

$$-C_{1}\alpha_{1}^{2} - C_{2}\alpha_{1}^{2} + C_{1}\alpha_{1}^{2} + C_{3}\beta_{1}^{2} = 0$$
  
$$\left(\alpha_{1}^{2} + \beta_{1}^{2}\right)C_{3} = 0, \ \alpha_{1}^{2} + \beta_{1}^{2} \neq 0 \implies C_{3} = 0.$$
(36)

Then from (I):

$$C_1 = -C_3 = 0$$
  

$$C_1 = C_3 = 0.$$
 (37)

The function  $w_1$  is now

$$w_1 = \left(C_2 \sin \alpha_1 y + C_4 \sin \beta_2 y\right) \sin \frac{m\pi x}{a} \qquad (38)$$

where  $\alpha_1$  and  $\beta_1$  are given with (23).

The conditions (III) and (IV) at the end of plate (2) are for y = b, the deflection and the moment are also equal to zero.

**III condition** for plate (2):

$$w_2 = 0 \Big|_{v=b}$$
. (39)

IV condition:

$$D_2 \left( \frac{\partial^2 w_2}{\partial y^2} + v \frac{\partial^2 w_2}{\partial x^2} \right) \bigg|_{y=b} = 0.$$
 (40)

From the function  $w_2 = f_2(y)\sin\frac{m\pi x}{a}$ , we have:  $w_2 = (C_5 \operatorname{ch}\alpha_2 y + C_6 \operatorname{sh}\alpha_2 y + C_7 \cos\beta_2 y + C_8 \sin\beta_2 y) \sin\frac{m\pi x}{a}$   $\frac{\partial w_2}{\partial y} = (C_5\alpha_2 \operatorname{sh}\alpha_2 y + C_6\alpha_2 \operatorname{ch}\alpha_2 y - C_7\beta_2 \sin\beta_2 y + C_8\beta_2 \cos\beta_2 y) \sin\frac{m\pi x}{a}$  $\frac{\partial^2 w_2}{\partial y^2} = (C_5\alpha_2^2 \operatorname{ch}\alpha_2 y + C_6\alpha_2^2 \operatorname{sh}\alpha_2 y - C_7\beta_2 \sin\beta_2 y + C_8\beta_2 \cos\beta_2 y) \sin\frac{m\pi x}{a}$ 

$$-C_7\beta_2^2\cos\beta_2 y - C_8\beta_2^2\sin\beta_2 y\bigg)\sin\frac{m\pi x}{a}.$$
 (41)

From (III) we obtain:

$$w_{2} = f_{2}(b) \sin \frac{m\pi x}{a} \Big|_{y=b} = (C_{5} \operatorname{ch} \alpha_{2} b + C_{6} \operatorname{sh} \alpha_{2} b + C_{7} \cos \beta_{2} b + C_{8} \sin \beta_{2} b) \sin \frac{m\pi x}{a}$$
$$f_{2}(b) = 0 = C_{5} \operatorname{ch} \alpha_{2} b + C_{6} \operatorname{sh} \alpha_{2} b + C_{7} \cos \beta_{2} b + C_{8} \sin \beta_{2} b = 0$$
(42)

and from IV:

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$$\frac{\partial^2 w}{\partial y^2} + v \frac{\partial^2 w}{\partial x^2} \Big|_{y=b} = 0$$

$$f_2''(b) f_0(x) + v f_2(b) f_0''(x) = 0$$

$$w_2 = (C_5 \operatorname{ch} \alpha_2 y + C_6 \operatorname{sh} \alpha_2 y + C_7 \cos \beta_2 y + C_8 \sin \beta_2 y) \sin \frac{m\pi x}{a}$$

$$\frac{\partial w_2}{\partial y} = (C_5 \alpha_2 \operatorname{sh} \alpha_2 y + C_6 \alpha_2 \operatorname{ch} \alpha_2 y - C_7 \beta_2 \sin \beta_2 y + C_8 \beta_2 \cos \beta_2 y) \sin \frac{m\pi x}{a}$$

$$\frac{\partial^2 w_2}{\partial y^2} = (C_5 \alpha_2^2 \operatorname{ch} \alpha_2 y + C_6 \alpha_2^2 \operatorname{sh} \alpha_2 y - C_7 \beta_2^2 \cos \beta_2 y - C_8 \beta_2^2 \sin \beta_2 y) \sin \frac{m\pi x}{a}.$$
(43)

Condition IV gives us:

$$\left(C_5\alpha_2^2 \operatorname{ch}\alpha_2 b + C_6\alpha_2^2 \operatorname{sh}\alpha_2 b - -C_7\beta_2^2 \cos\beta_2 b - C_8\beta_2^2 \sin\beta_2 b\right) \cdot \sin\frac{m\pi x}{a} + \nu \left(\frac{m^2\pi^2}{a^2}\right) f_2(b) \sin\frac{m\pi x}{a} = 0.$$
(44)

Taking into account condition (III):  $f_2(b) = 0$ , condition (IV) is reduced to:

$$C_5 \alpha_2^2 \operatorname{ch} \alpha_2 b + C_6 \alpha_2^2 \operatorname{sh} \alpha_2 b - -C_7 \beta_2^2 \cos \beta_2 b - C_8 \beta_2^2 \sin \beta_2 b = 0.$$
(45)

When condition (III) is multiplied first by  $\alpha_2^2$  and then by  $\beta_2^2$ , and in combination with condition (IV), by addition and subtraction we get:

$$C_{5}\left(\alpha_{2}^{2}+\beta_{2}^{2}\right)\operatorname{ch}\alpha_{2}b+\left(\alpha_{2}^{2}+\beta_{2}^{2}\right)\operatorname{sh}\alpha_{2}b=0$$
  

$$\alpha_{2}^{2}+\beta_{2}^{2}\neq0\implies C_{5}\operatorname{ch}\alpha_{2}b+C_{6}\operatorname{sh}\alpha_{2}b=0$$
  

$$C_{6}=-C_{5}\frac{\operatorname{ch}\alpha_{2}b}{\operatorname{sh}\alpha_{2}b}.$$
(46)

- III· 
$$\alpha_2^2$$
 + IV:  
- $C_7\left(\alpha_2^2 + \beta_2^2\right)\cos\beta_2 b - C_8\left(\alpha_2^2 + \beta_2^2\right)\sin\beta_2 b = 0$   
 $C_8 = -C_7\frac{\cos\beta_2 b}{\sin\beta_2 b}.$  (47)

When introducing (46) and (47), from (41) we get

$$w_{2} = \left[ C_{5} \operatorname{sh}\alpha_{2}y - C_{5} \frac{\operatorname{ch}\alpha_{2}b}{\operatorname{sh}\alpha_{2}b} \operatorname{sh}\alpha_{2}y + C_{7} \cos\beta_{2}y - -C_{7} \frac{\cos\beta_{2}b}{\sin\beta_{2}b} \sin\beta_{2}y \right] \sin\frac{m\pi x}{a}$$
$$w_{2} = C_{5} \frac{\operatorname{sh}\alpha_{2}(b-y)}{\operatorname{sh}\alpha_{2}b} + C_{7} \frac{\sin\beta_{2}(b-y)}{\sin\alpha_{2}b}. \quad (48)$$

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Now, having functions  $w_1(x,y)$  and  $w_2(x,y)$ :

$$w_{1} = f_{1}(y)f_{0}(x) = (C_{2} \operatorname{sh}\alpha_{1}y + C_{4} \sin\beta_{2}y)f_{0}(x)$$
$$w_{2} = f_{2}(y)f_{0}(x) =$$
$$= \left(C_{5} \frac{\operatorname{ch}\alpha_{2}(b-y)}{\operatorname{sh}\alpha_{2}b} + C_{7} \frac{\sin\beta_{2}(b-y)}{\sin\alpha_{2}b}\right)f_{0}(x) .$$
(49)

Now, we use the condition at  $y = \eta$ . Along the edge  $y = \eta$ , deflections between (1) and (2) are equal

$$w_1(x, y)\Big|_{y=\eta} = w_2(x, y)\Big|_{y=\eta}.$$
 (50)

The condition (50) is reduced to:

$$f_{1}(y)\big|_{y=\eta} = f_{2}(y)\big|_{y=\eta}$$

$$C_{2} \operatorname{sh}\alpha_{1}\eta + C_{4} \operatorname{sin}\beta_{1}\eta =$$

$$C_{5} \frac{\operatorname{sh}\alpha_{2}(b-\eta)}{\operatorname{sh}\alpha_{2}b} + C_{7} \frac{\operatorname{sin}\beta_{2}(b-\eta)}{\operatorname{sin}\beta_{2}b}, \qquad (51)$$

or

$$C_{2} \operatorname{sh} \alpha_{1} \eta + C_{4} \operatorname{sin} \beta_{1} \eta - C_{5} \frac{\operatorname{sh} \alpha_{2} (b - \eta)}{\operatorname{sh} \alpha_{2} b} - C_{7} \frac{\operatorname{sin} \beta_{2} (b - \eta)}{\operatorname{sin} \beta_{2} b} = 0.$$
(52)

**Condition VI**: We need the derivatives of (49):

$$\frac{\partial w_1}{\partial y} = \left(C_2 \alpha_1 \operatorname{ch} \alpha_1 y + C_4 \beta_1 \cos \beta_1 y\right) \sin \frac{m\pi x}{a}$$
$$\frac{\partial^2 w_1}{\partial y^2} = \left(C_2 \alpha_1^2 \operatorname{sh} \alpha_1 y - C_4 \beta_1^2 \sin \beta_1 y\right) \sin \frac{m\pi x}{a}$$
$$\frac{\partial^3 w_1}{\partial y^3} = \left(C_2 \alpha_1^3 \operatorname{ch} \alpha_1 y - C_4 \beta_1^3 \cos \beta_1 y\right) \sin \frac{m\pi x}{a}$$
$$\frac{\partial w_2}{\partial y} = \left(-C_5 \alpha_2 \frac{\operatorname{ch}_2(b-y)}{\operatorname{sh} \alpha_2 b} - C_7 \beta_2 \frac{\cos \beta_2(b-y)}{\sin \beta_2 b}\right) \sin \frac{m\pi x}{a}$$
$$\frac{\partial^2 w_2}{\partial y^2} = \left(C_5 \alpha_2^2 \frac{\operatorname{sh} \alpha_2(b-y)}{\operatorname{sh} \alpha_2 b} - C_7 \beta_2^2 \frac{\sin \beta_2(b-y)}{\sin \beta_2 b}\right) \sin \frac{m\pi x}{a}$$
$$\frac{\partial^3 w_2}{\partial y^3} = \left(-C_5 \alpha_2^3 \frac{\operatorname{ch} \alpha_2(b-y)}{\operatorname{sh} \alpha_2 b} + C_7 \beta_2^3 \frac{\cos \beta_2(b-y)}{\sin \beta_2 b}\right) \sin \frac{m\pi x}{a}.$$
(53)

Condition VI is: the slope of plate (1) is equal to the slope of plate (2) for  $y = \eta$ :

$$\frac{\partial w_1(x,y)}{\partial y}\bigg|_{y=\eta} = \frac{\partial w_2(x,y)}{\partial y}\bigg|_{y=\eta},$$
 (54)

this means that using (53) we come to  $f'_1(\eta) = f'_2(\eta)$ , or

$$C_{2}\alpha_{1}\operatorname{ch}\alpha_{1}\eta + C_{4}\beta_{1}\cos\beta_{1}\eta + C_{5}\alpha_{2}\frac{\operatorname{ch}\alpha_{2}(b-\eta)}{\operatorname{sh}\alpha_{2}b} + C_{7}\beta_{2}\frac{\cos\beta_{2}(b-\eta)}{\sin\beta_{2}b} = 0.$$
 (55)

**Condition VII**: Along the line of discontinuity, the moment of plate (1) is equal to the moment of plate (2), for  $y = \eta$ :

$$D_{1}\left(\frac{\partial^{2}w_{1}}{\partial y^{2}} + v\frac{\partial^{2}w_{1}}{\partial x^{2}}\right)\Big|_{y=\eta} = D_{2}\left(\frac{\partial^{2}w_{2}}{\partial y^{2}} + v\frac{\partial^{2}w_{2}}{\partial x^{2}}\right)\Big|_{y=\eta}.$$
(56)

Using (53), and notation  $\frac{D_1}{D_2} = \psi$ , and having derivatives we get:

$$\psi \left[ f_1''(y) - v \frac{m^2 \pi^2}{a^2} f_1(y) \right] \sin \frac{m \pi x}{a} = \left[ f_2''(y) - v \frac{m^2 \pi^2}{a^2} f_2(y) \right] \sin \frac{m \pi x}{a}$$
(57)

or

$$\psi \left[ \left( \alpha_1^2 C_2 \operatorname{sh} \alpha_1 \eta - C_4 \beta_1^2 \operatorname{sin} \beta_1 \eta \right) - \frac{w^2 \pi^2}{a^2} (C_2 \operatorname{sh} \alpha_1 \eta + C_4 \operatorname{sin} \beta_1 \eta) \right] = \left[ C_5 \alpha_2^2 \frac{\operatorname{sh} \alpha_2 (b - \eta)}{\operatorname{sh} \alpha_2 b} - C_7 \beta_2^2 \frac{\operatorname{sin} \beta_2 (b - \eta)}{\operatorname{sin} \beta_2 b} \right] - \frac{w^2 \pi^2}{a^2} \left[ C_5 \frac{\operatorname{sh} \alpha_2 (b - \eta)}{\operatorname{sh} \alpha_2 b} - C_7 \frac{\operatorname{sin} \beta_2 (b - \eta)}{\operatorname{sin} \beta_2 b} \right].$$
(58)

The final form is:

$$C_{2}\psi\left(\alpha_{1}^{2}-\nu\frac{m^{2}\pi^{2}}{a^{2}}\right) \mathrm{sh}\alpha_{1}\eta - C_{4}\psi\left(\beta_{1}^{2}+\nu\frac{m^{2}\pi^{2}}{a^{2}}\sin\beta_{1}\eta\right) - \\ -C_{5}\left(\alpha_{2}^{2}-\nu\frac{m^{2}\pi^{2}}{a^{2}}\right) \frac{\mathrm{sh}\alpha_{2}(b-\eta)}{\mathrm{sh}\alpha_{2}b} + \\ +C_{7}\left(\beta_{2}^{2}+\nu\frac{m^{2}\pi^{2}}{a^{2}}\right) \frac{\sin\beta_{2}(b-\eta)}{\sin\beta_{2}b}.$$
 (59)

**Condition VIII**: Sharing forces along the contact of plate (1) and (2), for  $y = \eta$  must be equal:

$$D_{1}\left[\frac{\partial^{3} w_{1}}{\partial y^{3}} + (2-\nu)\frac{\partial^{3} w_{1}}{\partial x^{2} \partial y}\right]\Big|_{y=\eta} = D_{2}\left[\frac{\partial^{3} w_{2}}{\partial y^{3}} + (2-\nu)\frac{\partial^{3} w_{3}}{\partial x^{2} \partial y}\right]\Big|_{y=\eta}$$
(60)

which can be written, introducing  $\psi$ , in the form

$$\psi \left[ f_1'''(y) - (2 - \nu) \frac{m^2 \pi^2}{a^2} f_1'(y) \right] \bigg|_{y=\eta} \sin \frac{m\pi x}{a} = \\ = \left[ f_2'''(y) - (2 - \nu) \frac{m^2 \pi^2}{a^2} f_2'(y) \right] \sin \frac{m\pi x}{a} \quad (61)$$

or, using necessary derivatives (16), for (60), we obtain:

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$$\begin{aligned}
& \psi \left\{ \left[ C_2 \alpha_1^3 \operatorname{ch} \alpha_1 \eta - C_4 \beta_1^3 \cos \beta_1 \eta \right] - (2 - \nu) \frac{m^2 \pi^2}{a^2} \cdot \left[ \alpha_1 C_2 \operatorname{ch} \alpha_1 \eta + \beta_1 C_4 \cos \beta_1 \eta \right] \right\} \sin \frac{m \pi x}{a} = \\
&= \left\{ \left[ -C_5 \alpha_2^3 \frac{\operatorname{ch} \alpha_2 (b - \eta)}{\operatorname{sh} \alpha_2 b} + C_7 \beta_2^3 \frac{\cos \beta_2 (b - \eta)}{\sin \beta_2 b} \right] - (2 - \nu) \cdot \left[ \cdot \frac{m^2 \pi^2}{a^2} \left[ -C_5 \alpha_2 \frac{\operatorname{ch} \alpha_2 (b - \eta)}{\operatorname{sh} \alpha_2 \eta} - C_7 \beta_2 \frac{\cos \beta_2 (b - \eta)}{\sin \beta_2 b} \right] \right\} \cdot \\
&\quad \cdot \sin \frac{m \pi x}{a} .
\end{aligned}$$
(62)

Form (62) can be written

$$\psi \left\{ C_2 \left[ \alpha_1^3 - (2-\nu)\alpha_1 \frac{m^2 \pi^2}{a^2} \right] \operatorname{ch} \alpha_1 \eta - - C_4 \left[ \beta_1^3 + (2-\nu)\beta_1 \frac{m^2 \pi^2}{a^2} \right] \cos \beta_1 \eta \right\} = -C_5 \left[ \alpha_2^3 - (2-\nu)\alpha_2 \frac{m^2 \pi^2}{a^2} \right] \frac{\operatorname{ch} \alpha_2 (b-\eta)}{\operatorname{sh} \alpha_2 b} + + C_7 \left[ \beta_2^3 + (2-\nu)\beta_2 \frac{m^2 \pi^2}{a^2} \right] \frac{\cos \beta_2 (b-\eta)}{\sin \beta_2 b}$$
(63)

or

$$\psi C_{2} \left[ \alpha_{1}^{3} - (2 - \nu) \alpha_{1} \frac{m^{2} \pi^{2}}{a^{2}} \right] \operatorname{ch} \alpha_{1} \eta - -\psi C_{4} \left[ \beta_{1}^{3} + (2 - \nu) \beta_{1} \frac{m^{2} \pi^{2}}{a^{2}} \right] \operatorname{cos} \beta_{1} \eta + + C_{5} \left[ \alpha_{2}^{3} - (2 - \nu) \alpha_{2} \frac{m^{2} \pi^{2}}{a^{2}} \right] \frac{\operatorname{ch} \alpha_{2} (b - \eta)}{\operatorname{sh} \alpha_{2} b} - - C_{7} \left[ \beta_{2}^{3} + (2 - \nu) \beta_{2} \frac{m^{2} \pi^{2}}{a^{2}} \right] \frac{\operatorname{cos} \beta_{2} (b - \eta)}{\operatorname{sin} \beta_{2} b} = 0. (64)$$

We have obtained 4 homogeneous algebraic equations with unknown constants  $C_2$ ,  $C_4$ ,  $C_5$  and  $C_7$ . As those systems have trivial solution when all constants are equal to zero, the solution could be obtained only in the case when the determinant of the system (65) is equal to zero.

# 3. CONCLUSION

The determinant (65) has given solutions for critical coefficient *k* for different ratios of a/b (Fig. 3), for 3 cases: m = 1, 2 and 3. Minimum value of buckling force is obtained for m = 1 and a/b = 0.8 using the relation

$$k_{1} = \frac{(N_{x})_{\text{cri}}}{D_{1}} \frac{b^{2}}{\pi^{2}} = \frac{(\sigma_{x})_{\text{cri}} h_{1} b^{2}}{D_{1} \pi^{2}}$$
(66)

	$C_2$	$C_4$	$C_{5}$	$C_7$	
V	chα <sub>i</sub> η	sinβη	$\frac{\sinh \alpha_2(b-\eta)}{\sinh \alpha_2 b}$	$\frac{\sin\beta_2(b-\eta)}{\sin\beta_2 b}$	
М	$lpha$ ch $lpha\eta$	$\beta \cos \beta \eta$	$\alpha_2 \frac{\mathrm{ch}\alpha_2(b-\eta)}{\mathrm{sh}\alpha_2 b}$	$\beta_2 \frac{\cos\beta_2(b-\eta)}{\sin\beta_2 b}$	=0
VII	$\psi\left(\alpha_1^2 - v\frac{m^2\pi^2}{a^2}\right)$ shq $\eta$	$-\psi\left(\beta_1^2 + v\frac{m^2\pi^2}{a^2}\right)\sin\beta_1\eta$	$\left(\alpha_2^2 - \nu \frac{m^2 \pi^2}{a^2}\right) \frac{\operatorname{sha}_2(b - \eta)}{\operatorname{sha}_2 b}$	$\left(\beta_2^2 - \nu \frac{m^2 \pi^2}{a^2} \frac{\sin \beta_2(b-\eta)}{\sin \beta_2 b}\right)$	
VIII	$\psi \left[ \alpha_1^3 - \alpha_1 (2 - \nu) \frac{m^2 \pi^2}{a^2} \right] $ chain	$-\psi\left[\beta_{1}^{3}-\beta_{1}(2-\nu)\frac{m^{2}\pi^{2}}{a^{2}}\cos \alpha_{1}\eta\right]$	$\left[\alpha_2^3 - \alpha_2(2-\nu)\frac{m^2\pi^2}{a^2}\right]\frac{\mathrm{ch}\alpha_2(b-\eta)}{\mathrm{sh}\alpha_2 b}$	$-\left[\beta_2^3 + \beta_2(2-\nu)\frac{m^2\pi^2}{a^2}\frac{\cos\beta_2(b-\eta)}{\sin\beta_2 b}\right]$	

$$\frac{C_2}{c_1} \qquad C_4 \qquad C_5 \qquad C_7$$

$$\frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_2}{c_2} \qquad C_7$$

$$\frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_1}{c_1} \qquad \frac{c_2}{c_2} \qquad \frac{c_2}{c_2} \qquad \frac{c_1}{c_1} \qquad \frac{c_2}{c_2} \qquad \frac{c_1}{c_2} \qquad \frac{c_1}{c_1} \qquad \frac{c_2}{c_2} \qquad \frac{c_1}{c_2} \qquad \frac{c_$$

or the critical stress in the plate (1)

$$(\sigma_x)_{\rm cri} = \frac{D_1 \pi^2 k_1}{h_1 b^2}, \qquad (67)$$



Figure 3. Final solution for critical force as a function of (a/b)

# REFERENCES

- Илношин, А.А.: Устоичивост пластинок и оболочек за пределами упругости, Прикладнаја математика и механика, Vol. 8, No. 4, pp. 337-360, 1944.
- [2] Stowell, E.Z.: A Unified Theory of Plastic Buckling of Colomns and Plates, NACA Technical Note:

1556, National Advisory Committee for Aeronautics, Washington, 1948.

- [3] Kollbrunner, C.F. and Meister, M: *Knicken*, Springer Verlag, Berlin, 1958.
- [4] Волмир, А.С.: Устоичивост деформируемих систем, Наука, Moscow, 1967.
- [5] Dunjić, M.M.: Stability of Constructiv and Material Anisotropic Plates Subjected to Thermomechanical Load, Faculty of Mechanical Engineering, University of Belgrade, Belgrade, 2010, (in Serbian).

## ПРОБЛЕМ СТАБИЛНОСТИ СТЕПЕНАСТИХ ПЛОЧА У ПЛАСТИЧНОЈ ОБЛАСТИ

#### Момчило Дуњић

У области стабилности правоугаоних степенастих плоча, у последње време, обрађени су многи проблеми у еластичној области. Та проблематика још није обрађивана у пластичној области, било у области малих еласто-пластичних деформација, било у области течења. У овом раду се говори о проблему губитка стабилности степенасте правоугаоне плоча (са две дебљине) у пластичној зони деформисања, оптерећене притискујућом силом у равни плоче која делује дуж ивица по којима се мења дебљина.